| N | $=$ the total number of components in a container; N also denotes the outermost component |
| :---: | :---: |
| n | = a specific component when numbered from inside out; i.e., $\mathrm{n}=1,2, \ldots, \mathrm{~N}$ |
| $\mathrm{r}_{\mathrm{n}}$ | = outside radius of component $n$, inches |
| $\mathrm{r}_{\mathrm{n}-1}$ | $=$ inside radius of component n , inches |
| ro | = bore radius of container, inches |
| ${ }^{\text {r }}$ N | $=$ outer radius of container, inches |
| $\mathrm{k}_{\mathrm{n}}$ | = wall ratio of component $\mathrm{n}, \mathrm{k}_{\mathrm{n}}=\mathrm{r}_{\mathrm{n}} / \mathrm{r}_{\mathrm{n}-1}$ |
| K | = over-all wall ratio of container, $\mathrm{K}=\mathrm{r}_{\mathrm{N}} / \mathrm{r}_{\mathrm{O}}$ |
| K ${ }^{\prime}$ | $=$ wall ratio of inner part of ring-fluid-segment container, $K^{\prime}=r_{3} / r_{\text {o }}$ |
| $E_{n}$ | $=$ modulus of elasticity of component n , psi |
| $\mathrm{p}_{\mathrm{n}}$ | $=$ pressure acting on component n at $\mathrm{r}_{\mathrm{n}}$ when $\mathrm{p} \neq 0$, psi |
| $\mathrm{p}_{\mathrm{n}-1}$ | = pressure acting on component n at $\mathrm{r}_{\mathrm{n}-1}$ when $\mathrm{p} \neq 0$, psi |
| p | = bore pressure, $\mathrm{psi}, \mathrm{p}_{\mathrm{o}}=\mathrm{p}$ |
| $\mathrm{q}_{\mathrm{n}}$ | $=$ residual interface pressure acting on component n at $\mathrm{r}_{\mathrm{n}}$ when $\mathrm{p}=0$, psi |
| $\mathrm{q}_{\mathrm{n}-1}$ | $=$ residual interface pressure acting on component $n$ at $r_{n-1}$ when $\mathrm{p}=0$, psi |
| S | = shear stress, psi |
| $\mathrm{S}_{\mathrm{r}}$ | = semi-range in shear stress for a cycle of bore pressure, psi |
| $S_{m}$ | $=$ mean shear stress for a cycle of bore pressure, psi |
| $\mathrm{S}_{\text {min }}$ | $=$ minimum shear stress during a cycle of bore pressure, psi |
| $S_{\text {max }}$ | $=$ maximum shear stress during a cycle of bore pressure, psi |
| $\sigma$ | $=$ design tensile stress of ductile steel, psi ( $\sigma \leqq$ ultimate tensile strength) |
| ${ }^{\sigma} 1$ | ```= design tensile stress of high-strength steel, psi ( }\mp@subsup{\sigma}{1}{}\leqq\mathrm{ ultimate tensile strength)``` |
| $(\sigma){ }_{r}$ | = semirange in tensile stress for a cycle of bore pressure, psi |
| $(\sigma)_{\mathrm{m}}$ | $=$ mean tensile stress for a cycle of bore pressure, psi |

## LIST OF SYMBOLS

 (Continued)$(\sigma)_{\min }=$ minimum tensile stress during a cycle of bore pressure, psi
$(\sigma)_{\max }=$ maximum tensile stress during a cycle of bore pressure, psi
$\sigma_{r} \quad=$ radial stress, psi
$\sigma_{\theta} \quad=$ circumferential stress, psi
$\sigma_{z} \quad=$ axial (longitudinal) stress, psi
$\alpha_{r} \quad=$ semirange stress parameter for high-strength steel, $\alpha_{r}=(\sigma)_{r} / \sigma_{1}$
$\alpha_{m} \quad=$ mean stress parameter for a high-strength steel, $\alpha_{m}=(\sigma)_{m} / \sigma_{1}$
$\mathrm{M}_{1} \quad=$ bending moment on ring segment
$\mathrm{M}_{2} \quad=$ bending moment on pin segment
u = radial displacement, inches
v = circumferential displacement, inches
$\nu \quad=$ Poisson's ratio
$\mathbf{r}, \theta, \mathbf{z}=$ cylindrical coordinates for radial, circumferential, and axial directions, respectively
$\Delta_{n} \quad=$ interference required (as manufactured) between cylinder, $n$, and cylinder, $n+1$, inches
$\Delta_{12}=$ interference required (as manufactured) between the liner, segments, and cylinder, 3 , of the ring-segment and ring-fluid-segment containers, inches

